



# **DYNAMIC ANALYSIS OF POWERTRAIN SYSTEMS – STATE OF THE ART AND FUTURE DEVELOPMENTS**

**Johannes Kepler University – Division of Technical Mechanics**  
Hans Irschik

**MAGNA POWERTRAIN – Engineering Center Steyr GmbH & Co KG**  
Bernhard Unger  
Helmut Dannbauer  
Harald Riener

**Junior Researcher**  
Wolfgang Witteveen

# Overview

Powertrain Simulation – State of the Art

Powertrain Simulation – Challenge: ‚Local Non-Linearities‘

General Considerations

Demonstrative Example: Bolted Joint

Powertrain Simulation – Tabular Summary

Future Strategies – MAGNA Powertrain Developments

# Powertrain Simulation: State of the Art

## Dominating simulation techniques

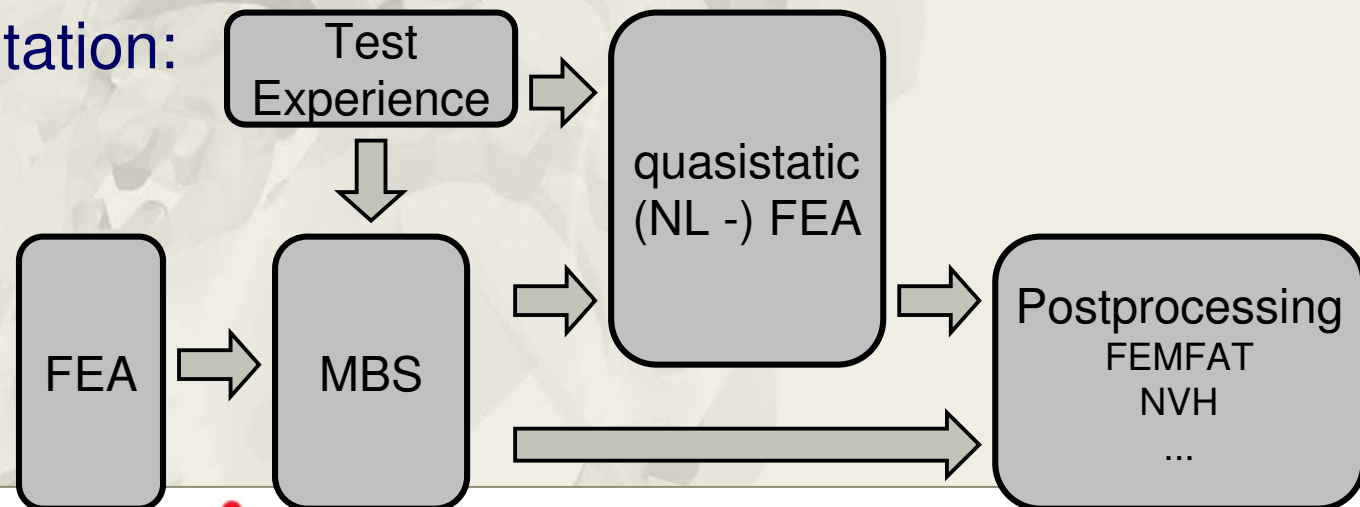
### (NL) Finite Element Analysis

- ✓ Non-linear static problems
- ✗ Dynamic simulation (time integration)

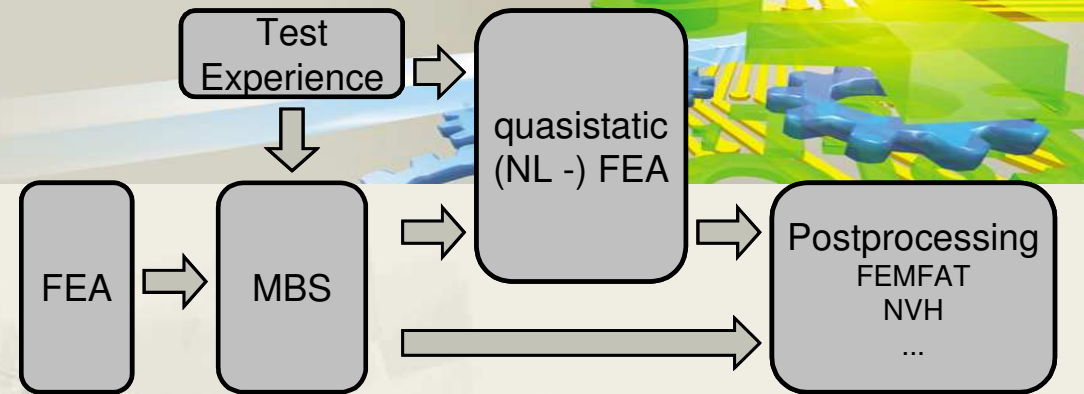
### Multi Body Simulation

- ✓ Dynamics of rigid bodies and monolithic linear elastic FE-Structures (modal representation).
- ✗ Dynamics of FE-Structures with (local) non-linear characteristic.

### Typical computation:



# Powertrain Simulation: State of the Art



## Good efficiency:

- Full dynamic of monolithic linear elastic FE-Structures (like crankshaft)

## Possibly acceptable efficiency:

- Transient computation of non-linear FE Structures
  - Not too much time steps (Which time steps?)
  - Linearized MBS has to deliver meaningful 'Internal forces' or 'global displacements'

} Significant limitation in terms of accuracy and process-automation

## Unsatisfying efficiency:

- Full dynamic of non-linear FE Structures and long time series
  - Economically not possible

# Powertrain Simulation: Challenge: ‚Local Non-Linearities‘

## General considerations

### Local non-linear characteristic

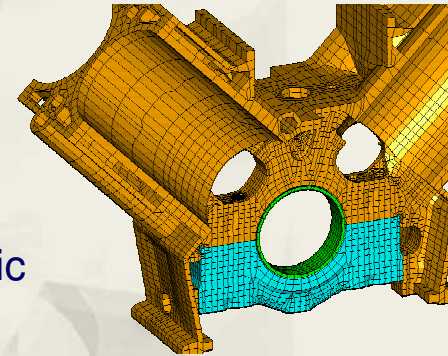
I.g.: Local non-linearities dominate the mechanical strength of a bearing chair.

Preload

Local non-elastic material characteristic

Pressed-in bearing shell

### significant local effects



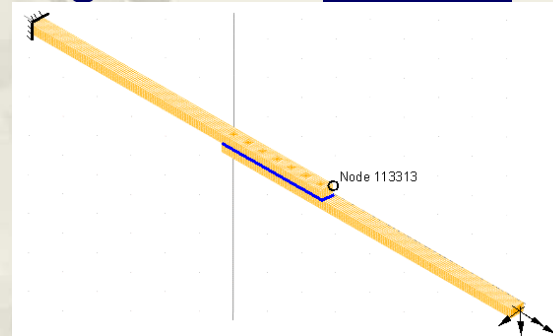
### Local non-linear characteristic

I.g.: Bolted Joints: Preload will influence

Global stiffness

Global damping

### significant global effects



# Powertrain Simulation: Challenge: ‚Local Non-Linearities‘

## Detailed example: Bolted Joints (Contact Problem) in MBS

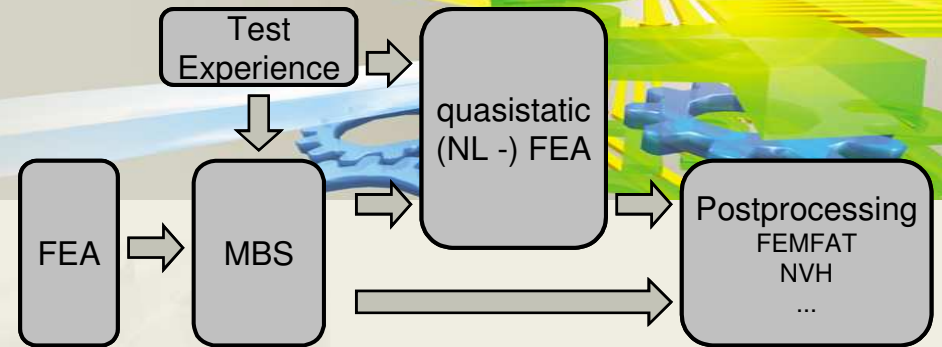
E.g.: Surface velocities of engine-transmission-unit. Joints main source of damping

Reality	MBS – modelling: State of the Art	Remarks
Stiffness non-linear function of preload and system state	Stiffness linearized	} Change of parameters Computation with updated FE model
Damping non-linear function of system state and some parameters	Damping linearized	
Local effect	Global, modal damping	• Damping values? • Results accuracy?
Damping is almost frequency independent	modal damping is frequency depended	High frequencies overdamped

What would be nice? : A physically meaningful, nonlinear consideration of the joints contact without losing the typical efficiency of MBS

# Powertrain Simulation: Tabular Summary

## State of the Art



	Computational effort (long time series)	Dynamic effects	Parameters physical relevance	Accuracy of Results	Typical field of application
Linear Structures	✓	✓	✓	✓	Crankshaft
Non-linear Structures – Linearized	✓	✓	?	?	Acoustics
Non-linear Structures – Internal Forces (NL-FEA)	✗	✗	✓	?	
Non-linear Structures – Substructure Analysis (NL-FEA)	✗	?	✓	?	

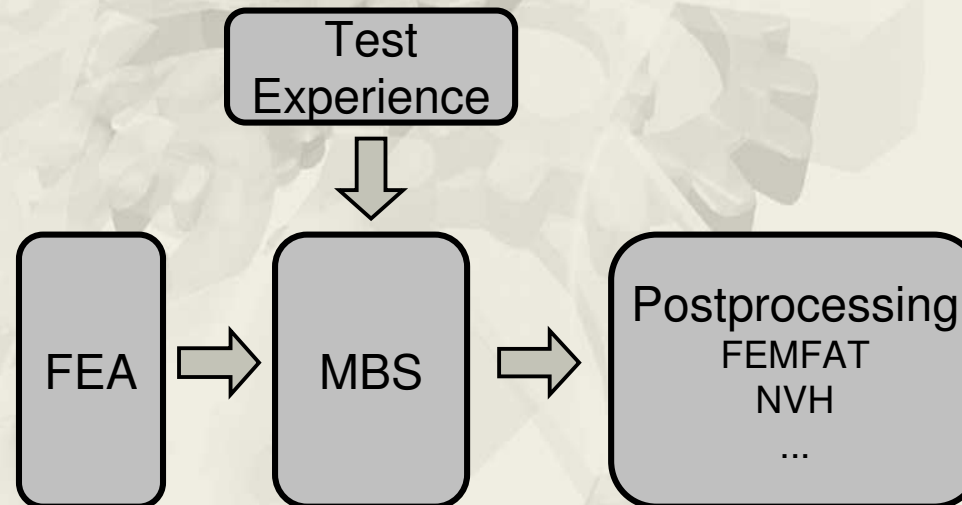
## What would be nice?

Non-linear Structures | ✓ | ✓ | ✓ | ✓ |

# Powertrain Simulation - Future Strategies:

Consideration of local non-linearities inside the MBS for  
acoustics (first step) and  
stress computation for lifetime prediction (second step)

Significant boundary condition is to provide the typical efficiency of MBS





# Powertrain Simulation - Future Strategies:

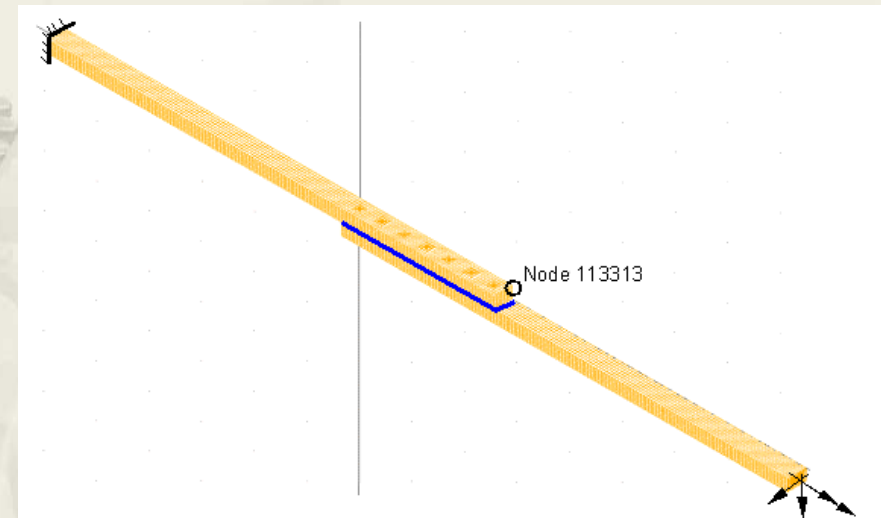
Development of MBS – ‚Plug-In‘ to regard non-linearities in MBS:

Demonstration example: Cantilever beam

- Two jointed substructures
- Both ends fixed
- Impulsive displacement excitation of one end
- Particular Node for evaluation

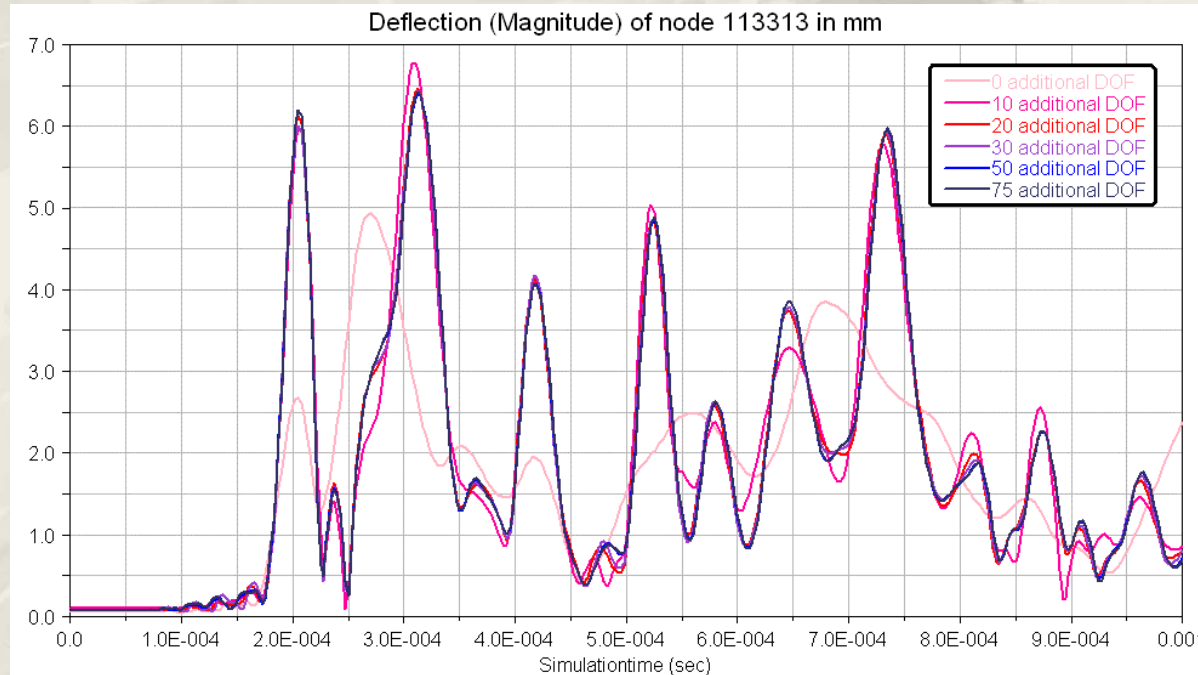
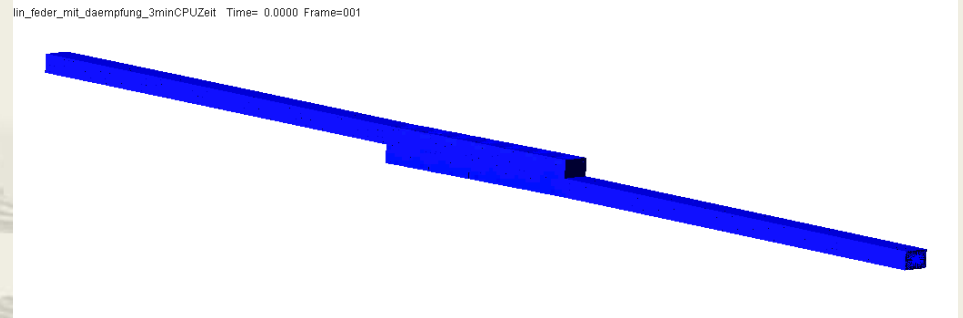
Goal:

- Full dynamic analysis inside a MBS software
- Consideration of contact
- Sufficient accuracy by significant save of computational time with respect to non linear FE computations (like ABAQUS)



# Powertrain Simulation - Future Strategies:

## Results:



- Unacceptable results in case of no additional DOF's for the MBS
- Excellent Results with 10, 20 or more additional DOF's for the MBS
- Entire Number of DOF's is 134 + ,additional DOF's' instead of several thousands which are necessary for a non-linear FE computation

# Powertrain Simulation - Future Strategies:

## Eye-Catcher: Local Damping

no\_damping Time= 0.0000 Equilibrium Frame=001



very\_high\_damping Time= 0.0000 Equilibrium Frame=001



# Powertrain Simulation - Future Strategies

## Summary

Application engineers challenge: Non-linearities

State of the Art:

Linearization (Acoustics)

MBS Non-linear FEA (Stress)

MAGNA Powertrain Development of MBS – ‚Plug In’ in order to regard the non-linearities in the MBS

Theoretical foundation elaborated

First examples promise a big potential

**Vision: Software package and parameters for solving contact problems inside the MBS**