



# Fatigue Life Estimation of Connecting Rod

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# Company Background

- Hero MotoCorp Ltd. is the world's largest manufacturer of two - wheelers, based in India.
- Hero MotoCorp has four manufacturing facilities based at Dharuhera, Gurgaon, Haridwar and Neemrana
- These plants together have a production capacity of 7.6 million 2-wheelers per year.



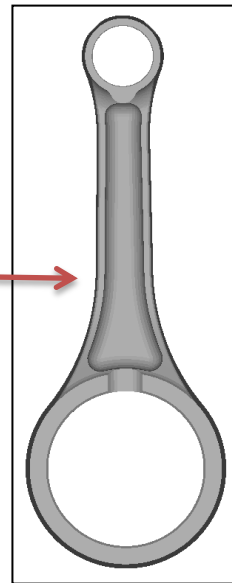
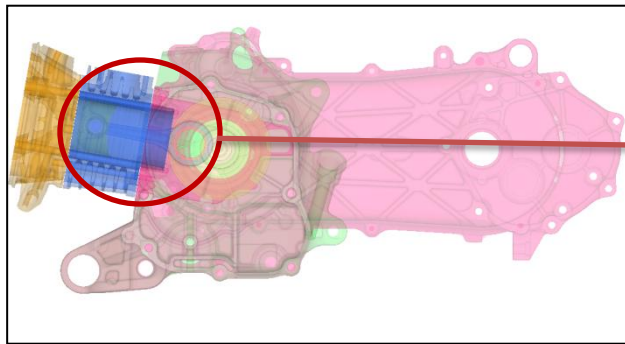
  
**Hero**  
**Garden  
Factory**  
Neemrana Complex  
*A state-of-the-art manufacturing plant*

# Agenda

- ✓ Objective
- ✓ Introduction
- ✓ Testing Method
- ✓ Fatigue Design Cycle
- ✓ Simulation Methodology
- ✓ Results
- ✓ Conclusion and Future Scope

# Objective

## Fatigue Life Estimation of Connecting Rod

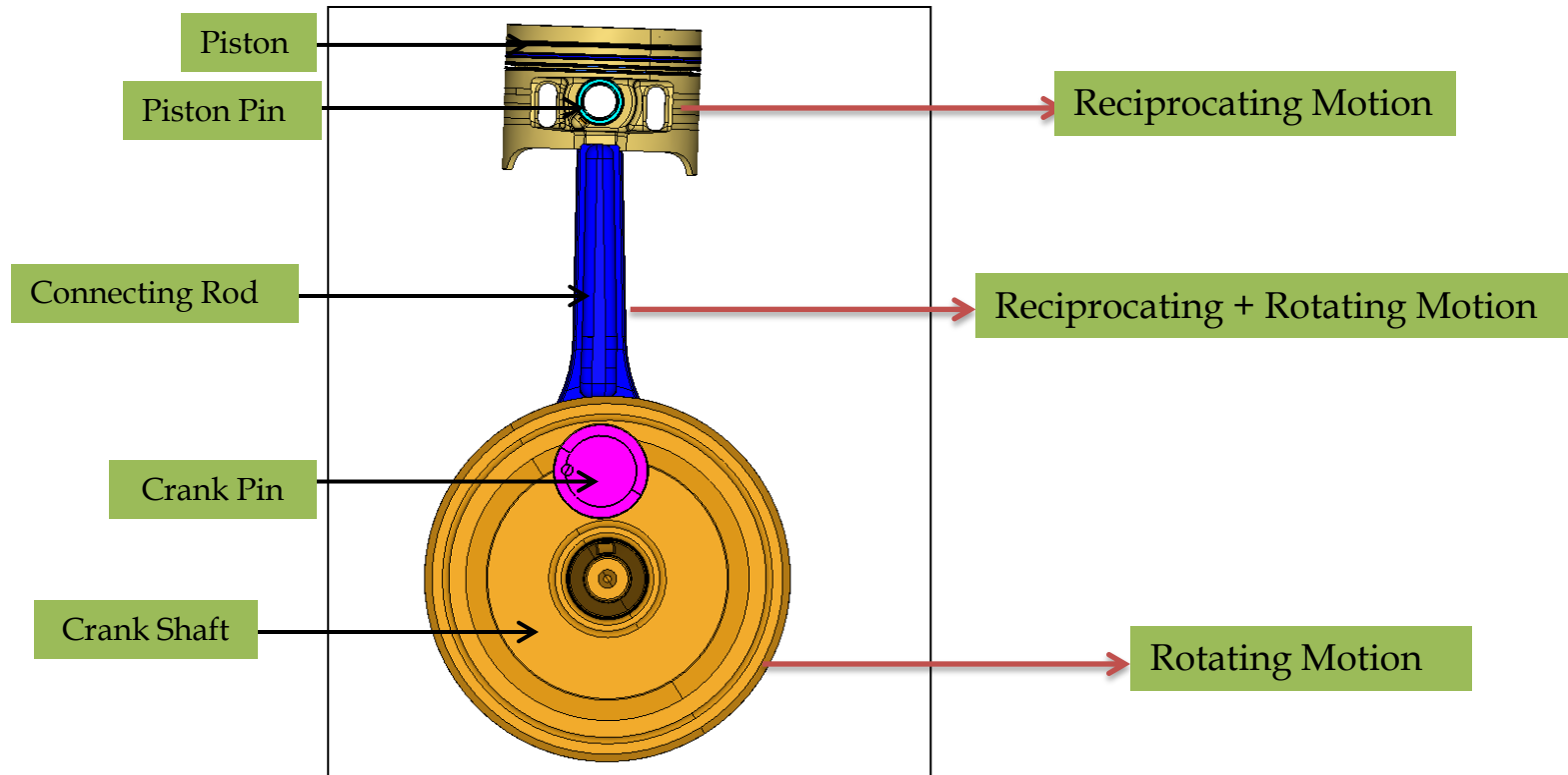


Connecting Rod

### Challenges

- ✓ Understand testing procedure
- ✓ Define fatigue cycle limit
- ✓ Define all load cases
- ✓ Define fatigue design cycle
- ✓ Standardize simulation process
- ✓ Fatigue life estimation through FEA simulation

# Introduction

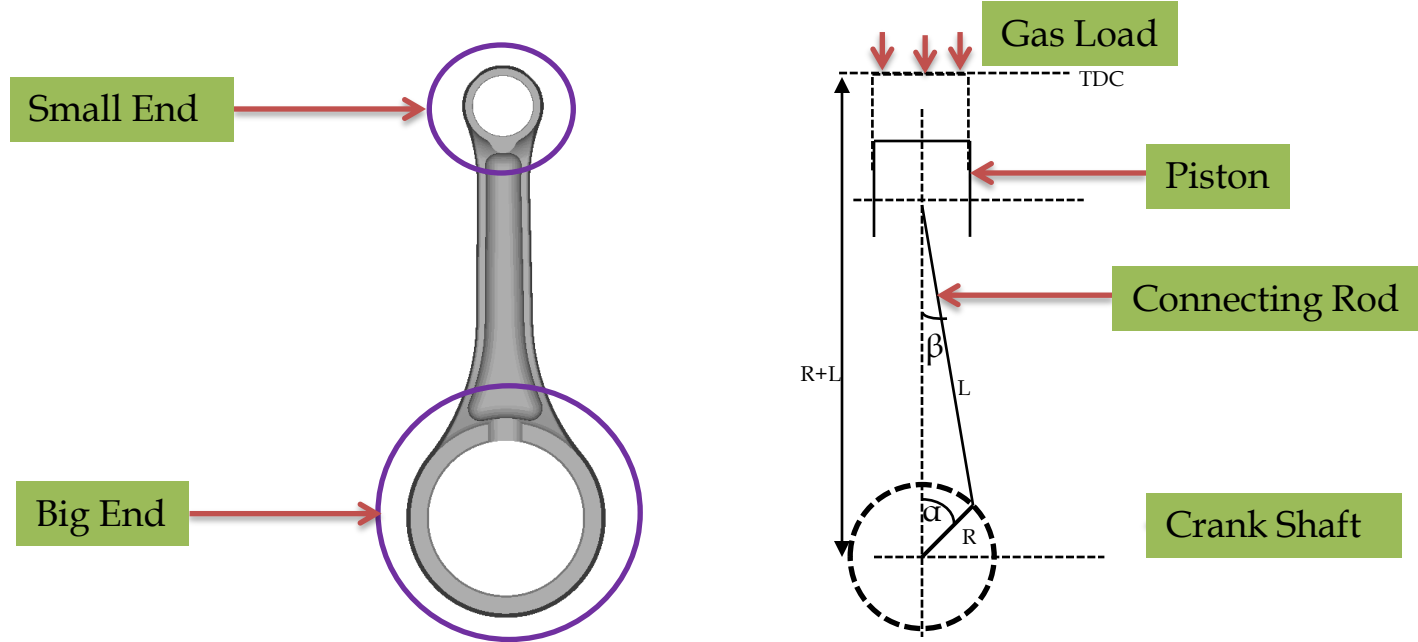


**Engine** : Source of power for Automobile, Convert chemical energy in to mechanical energy

**Crank-Train** : Convert Thermal energy of burned gases in to mechanical energy

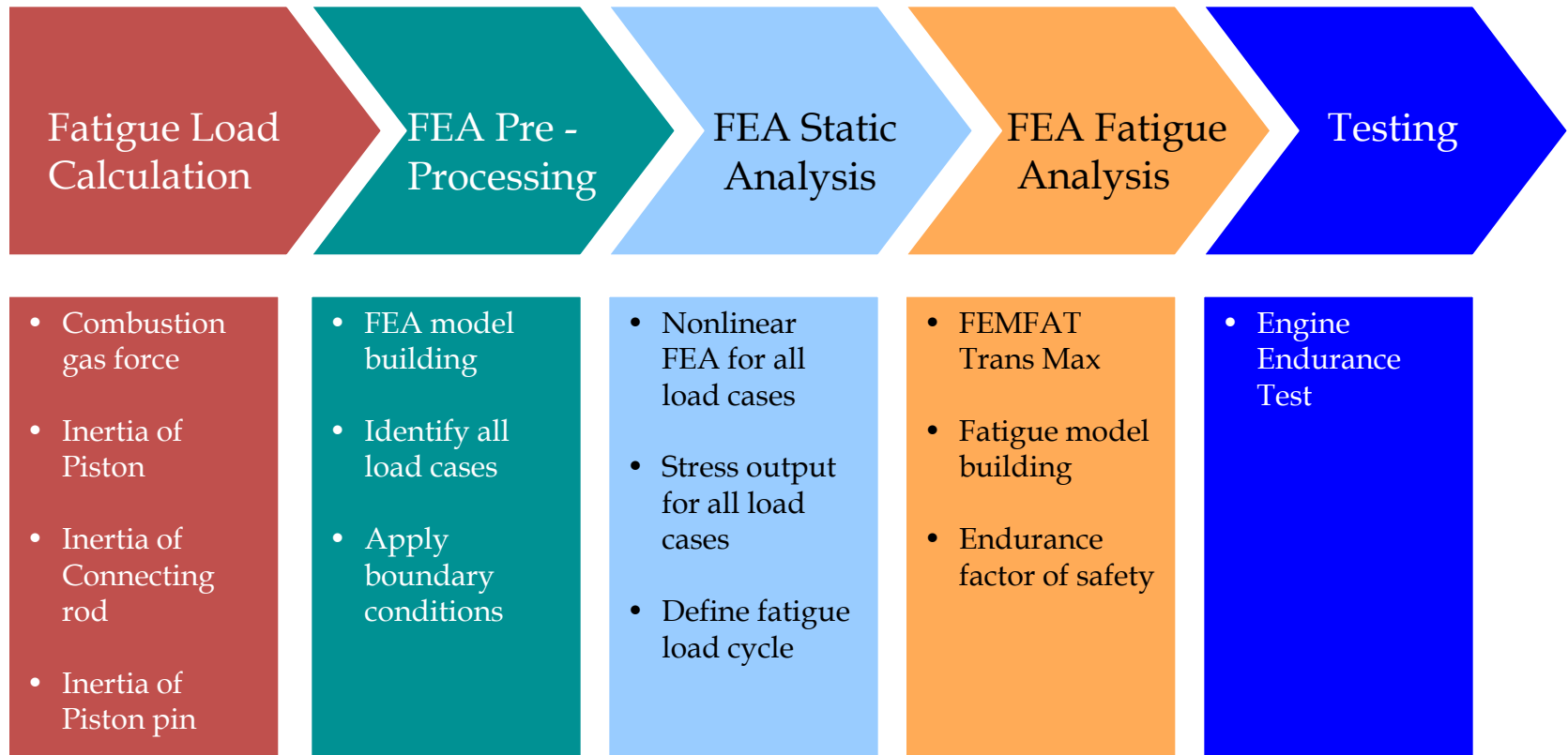
# Introduction

**Connecting Rod :** Convert reciprocating motion of piston in rotary motion of crankshaft

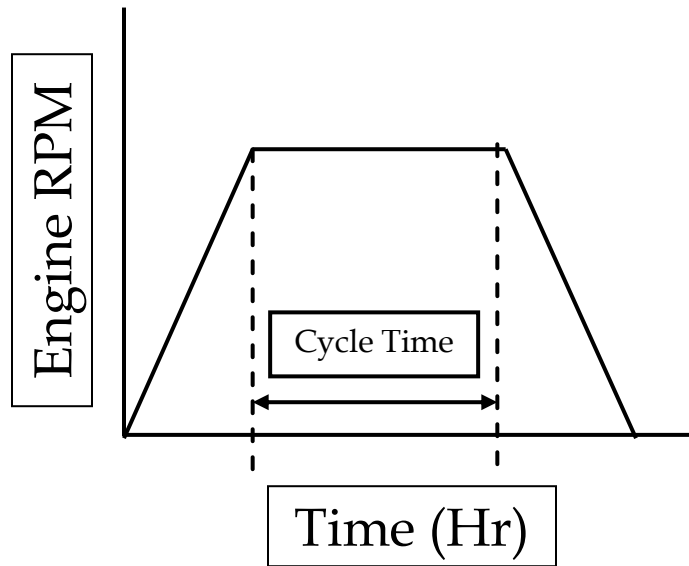


The connecting rod is one of the critical components of internal combustion engine. Crank Train dynamics generate continuous cyclic loading of reciprocating forces and rotating forces on connecting rod.

# Process Flow Chart



# Testing Methodology



Engine RPM =  $n$

Cycle Time =  $T$

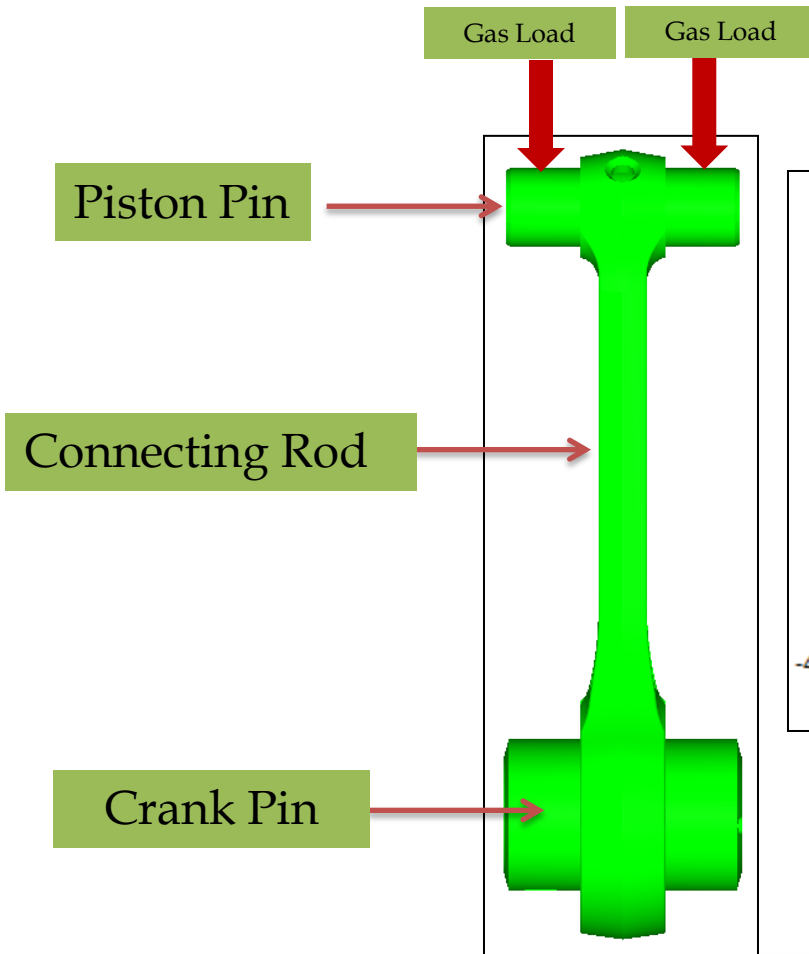
**Engine Endurance Test:** Put Engine on dynamometer at specified condition for defined cycle time

$$\text{No. of Fatigue Cycle } N = (n \times T \times 3600)/2$$

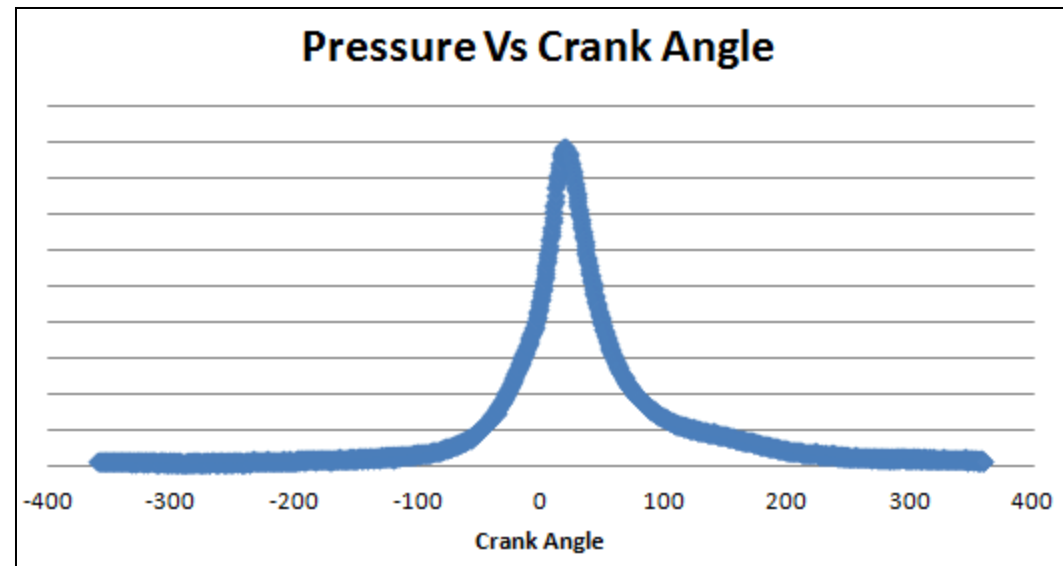


# Fatigue Cyclic Load

## Combustion Pressure Force

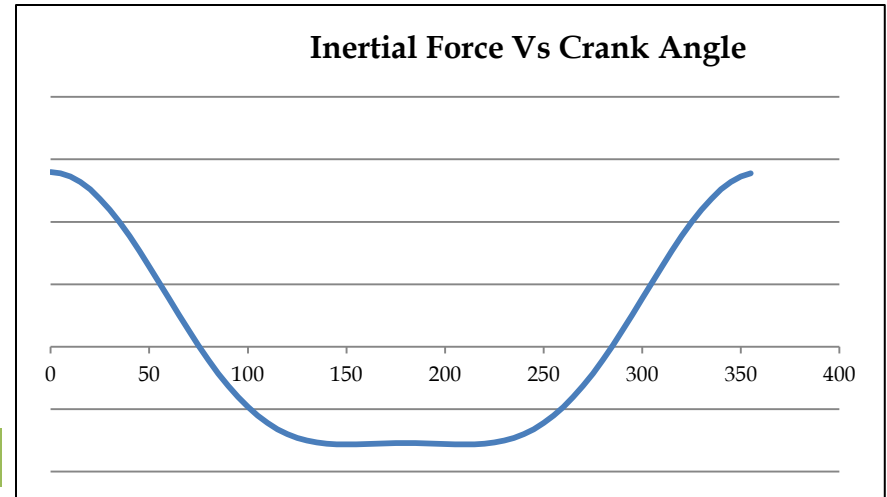
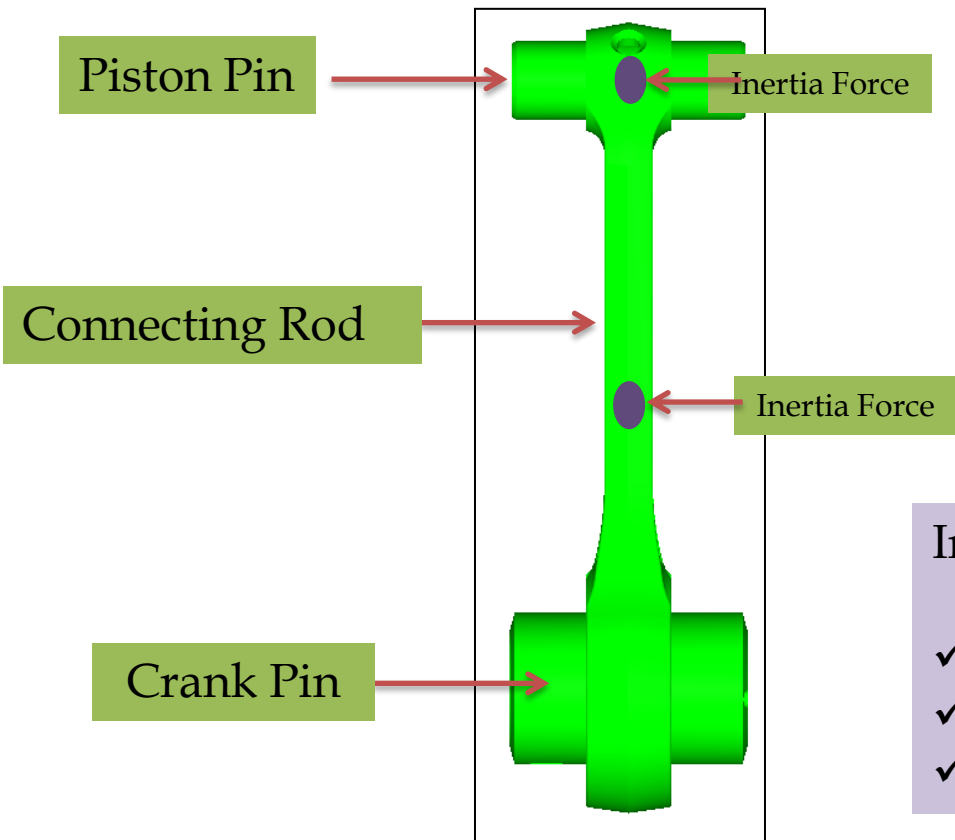


## Distributed Pressure Force



# Fatigue Cyclic Load

## Inertia Forces

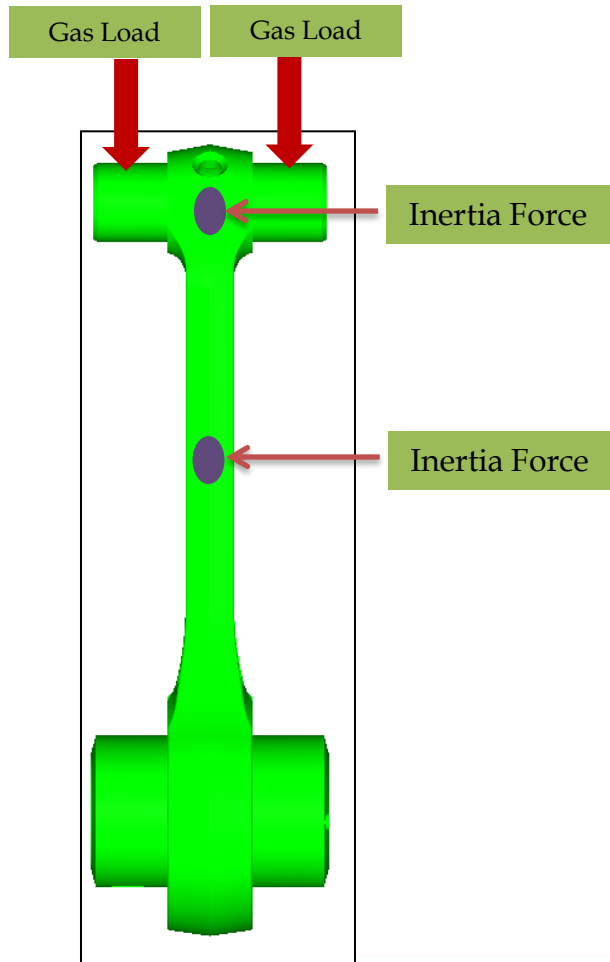


### Inertia Forces:

- ✓ Inertia Force of Piston
- ✓ Inertia Force of Connecting rod
- ✓ Inertia Force of Piston Pin

# Fatigue Cyclic Load

## Load Cases



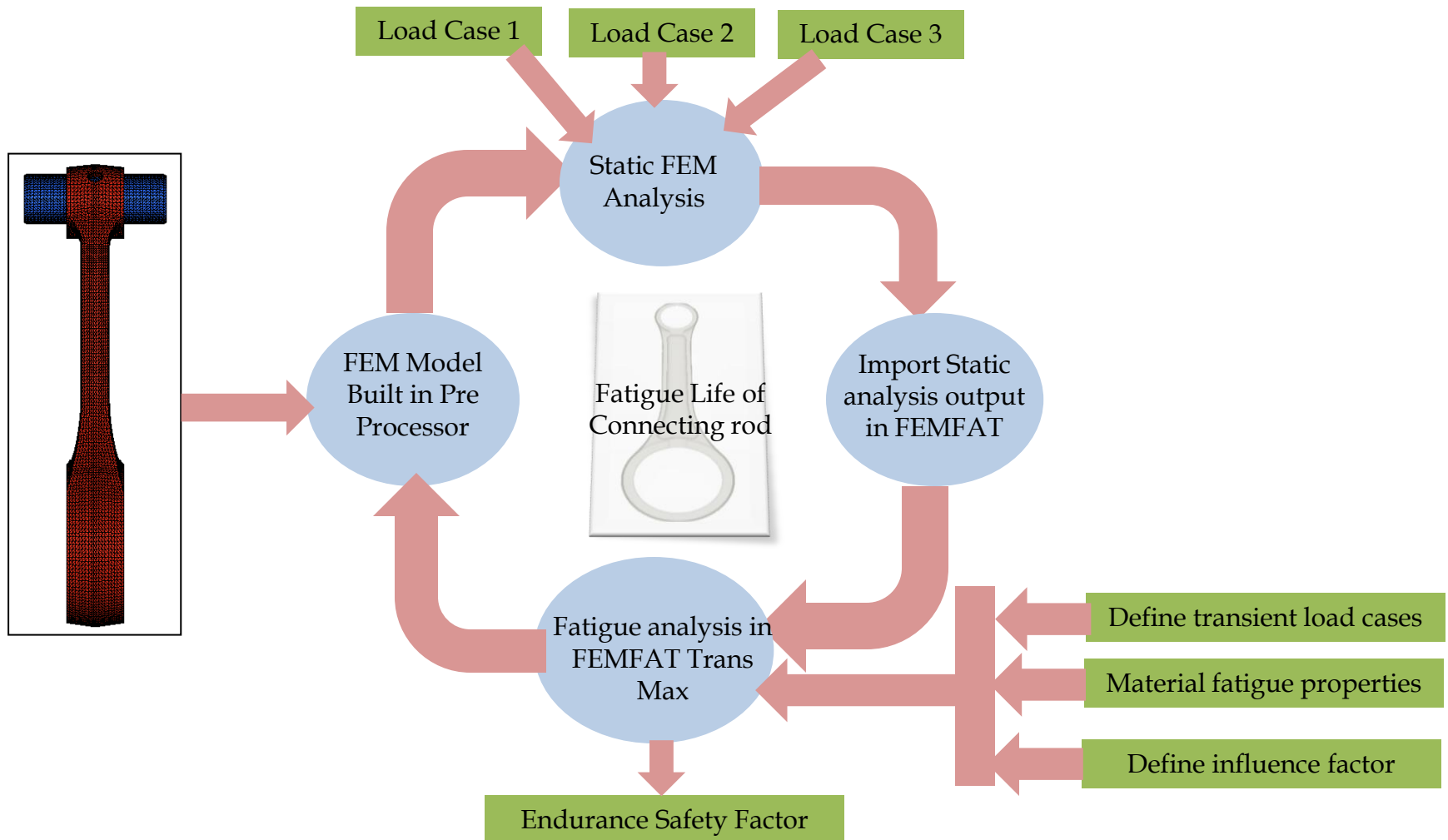
Engine RPM = N (Testing RPM )

Load Case 1 = Gas Force + Inertia Load

Load Case 2 = Inertia Load ↓

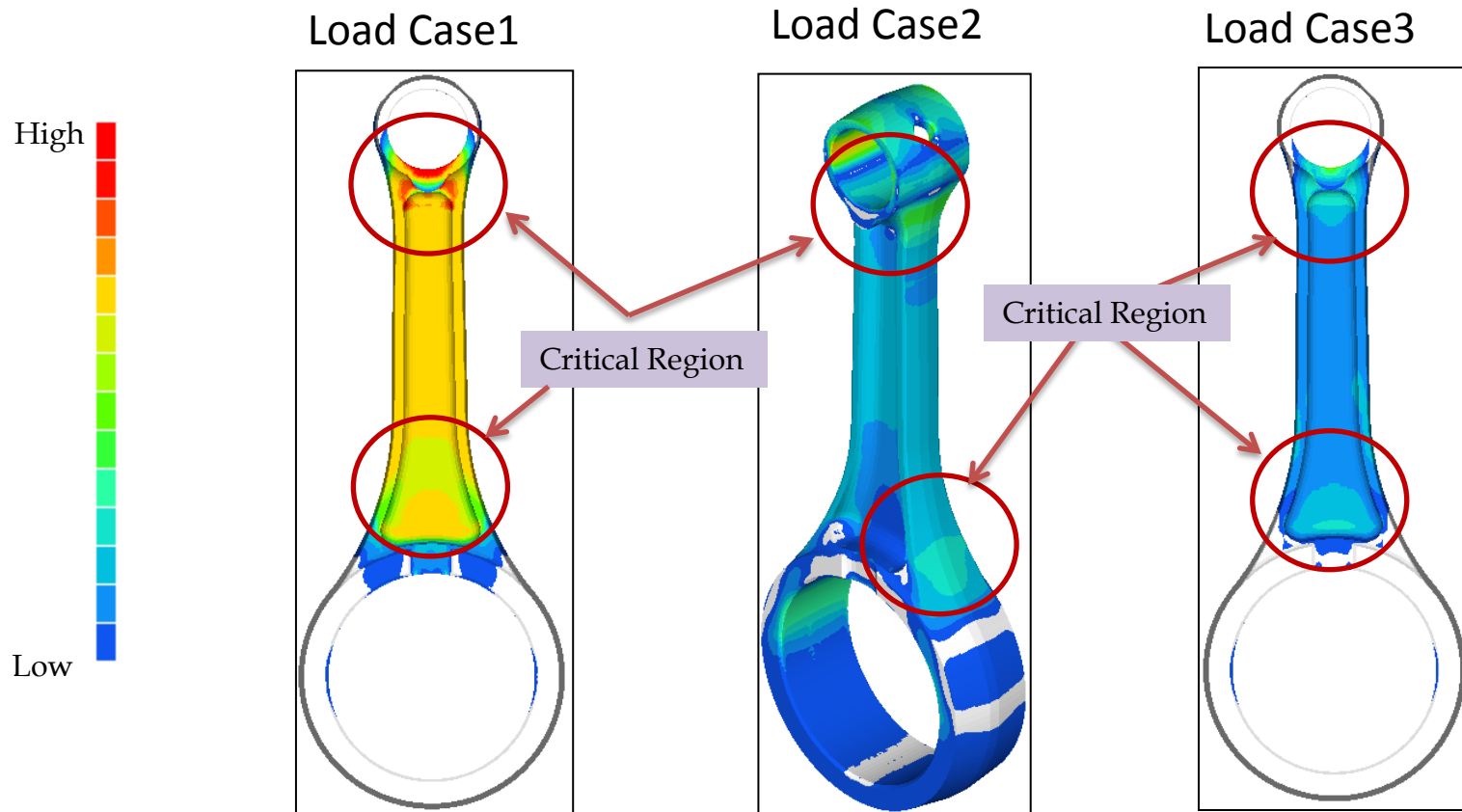
Load Case 3 = Inertia Load ↑

# Simulation Methodology



# Static Analysis

## Von Mises Stress

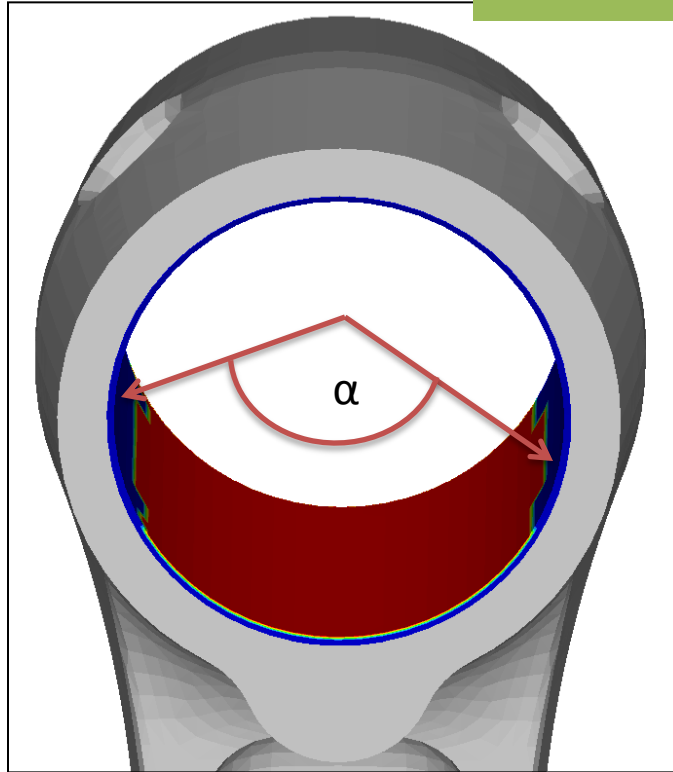


# Static Analysis

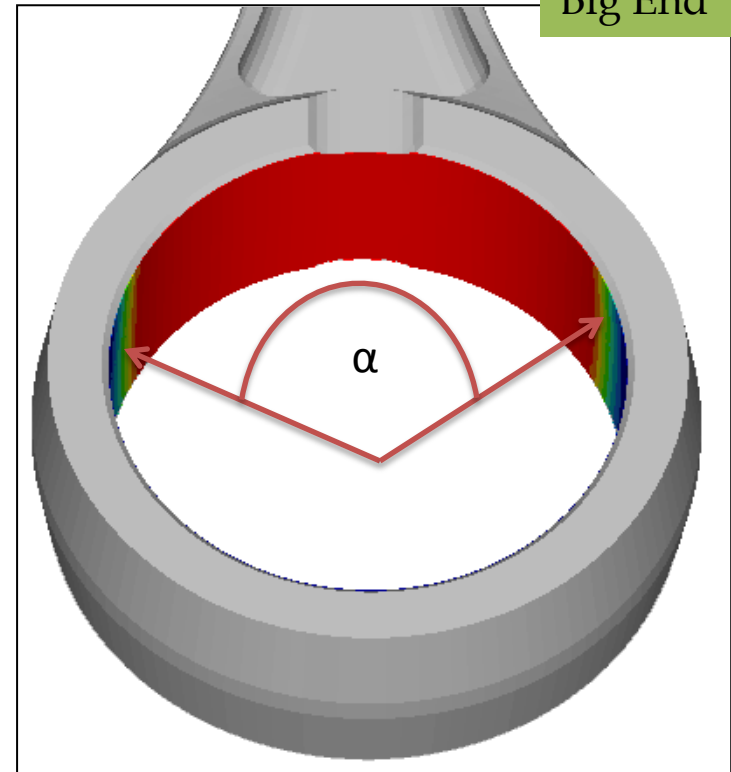
## Open Gap



Small End



Big End



Contact angle  $\alpha$  should be more than design specification

# Fatigue Analysis - FEMFAT Trans Max

## FEMFAT Trans Max Solution Parameter

Stress Gradient ON

Mean Stress ON

Surface Roughness ON

Mean (and Amplitude )  
Rearrangement ON

Modified Haigh Diagram ON

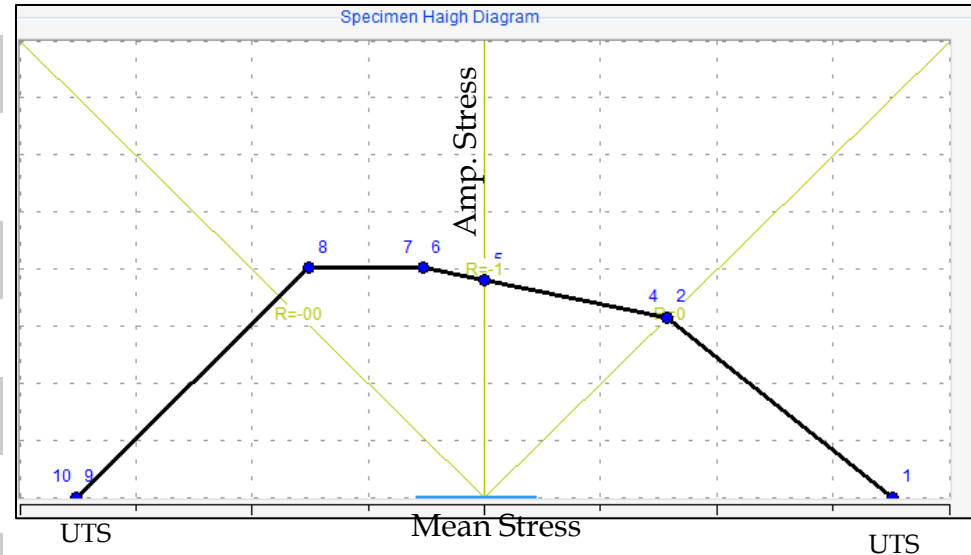
Statistical Influence ON

Temperature Influence OFF

Survival Probability 99.99%

Design Target Life N (Endurance Test)

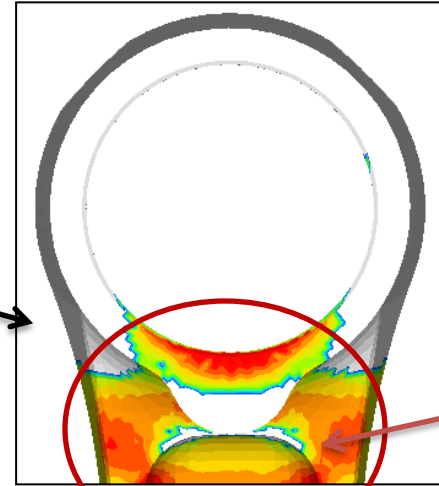
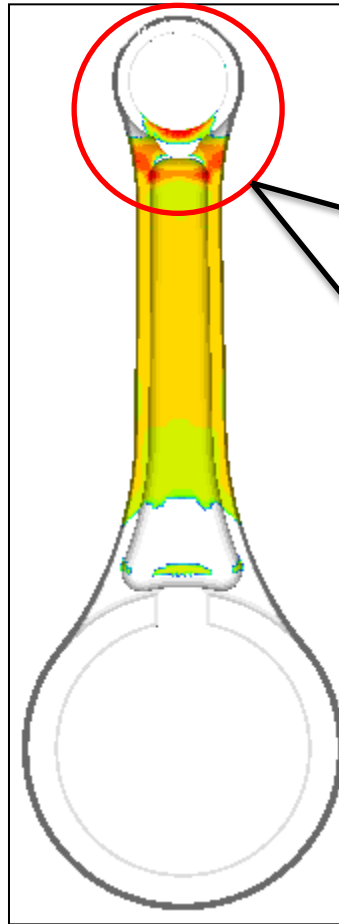
Analysis Type Endurance safety factor



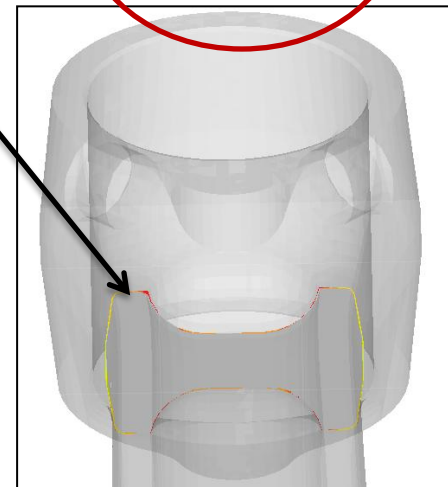
Typical Haigh Diagram of Hardened Steel material

# Fatigue Analysis - FEMFAT Trans Max

## Amplitude Stress



Critical Region

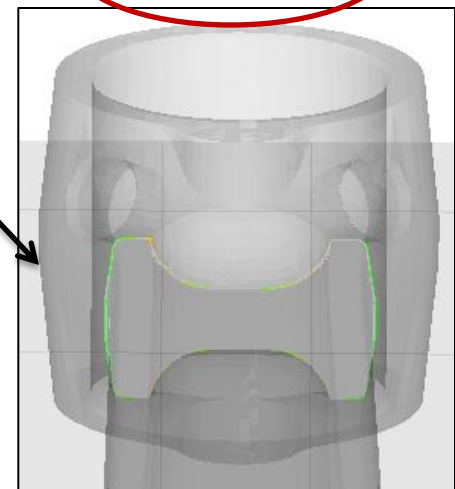
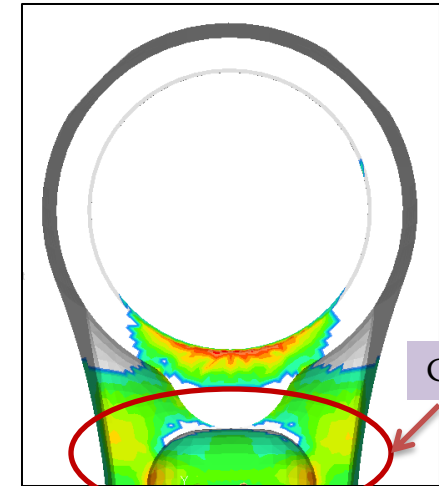
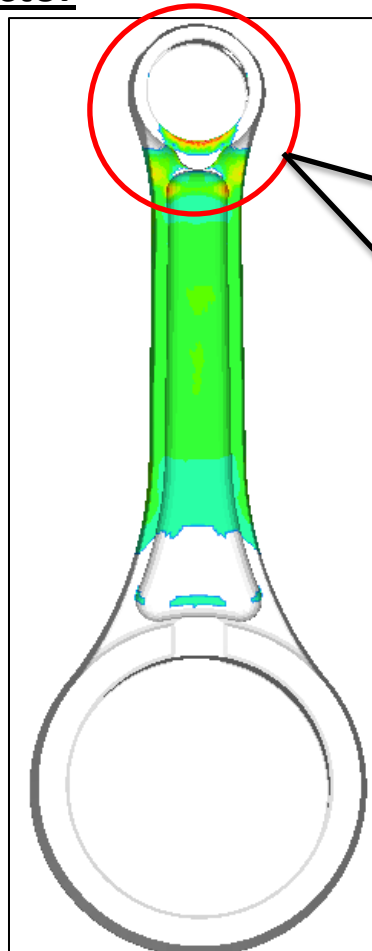


Amplitude Stress at critical Plane



# Fatigue Analysis - FEMFAT Trans Max

## Endurance Safety Factor



Minimum Life of Connecting rod is X times target life

Amplitude Stress at critical Plane

# Fatigue Analysis – FEMFAT Trans Max

## Conclusion

- ✓ Target fatigue life of connecting rod has been defined based on engine endurance test
- ✓ Fatigue cyclic load was calculated by crank train dynamics
- ✓ Static analysis of connecting rod shows that small end is more critical for durability
- ✓ FEMFAT Trans Max module was used for fatigue life estimation
- ✓ Fatigue result shows the small end region is critical for fatigue life

## Future Scope

- ✓ Component level testing can be develop for defined fatigue load cycle